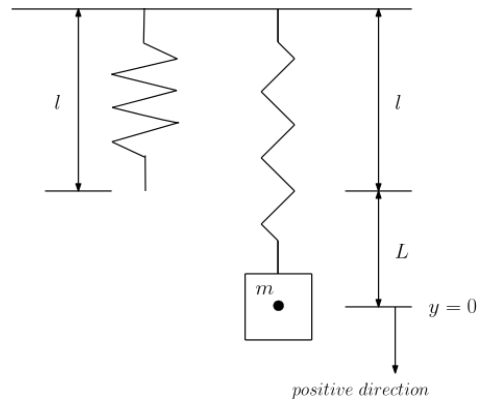


# THE MASS SPRING PROBLEM

Consider an unstretched spring of length  $l$ .



Now, attach an object of mass  $m$  to the spring. It will be stretched downward a length  $L$ . This stretched length will be our starting point for analysis. Let this point be defined as  $y = 0$  with a positive direction in the downward direction.

From Newton's second law we have  $F = m \cdot a$  or in our case  $F = m \cdot y''$ . What are the forces that can act on this object?

## 1. Gravity

The force of gravity will always act on this object and will be defined as

$$F_g = m \cdot g.$$

## 2. Spring

Hooke's law will govern the force on the spring.

$$F_s = -k \cdot (L + y)$$

## 3. Damping

Damping will resist any movement and will be proportional to the velocity of the object.

$$F_d = -\gamma \cdot y'$$

## 4. External forces

In this term you can put in any other forces acting on the object as a function of time.

$$F(t)$$

Now we add all of these forces into our Newton's Second Law equation.

$$m \cdot y'' = m \cdot g - k \cdot (L + y) - \gamma y' + F(t)$$

$$m \cdot y'' + \gamma y' + k \cdot y = m \cdot g - k \cdot L + F(t)$$

But, at equilibrium the system is at rest and  $m \cdot g = k \cdot L$ . This can be used to determine  $k$ . So we can rewrite our equation as

$$m \cdot y'' + \gamma y' + k \cdot y = F(t) .$$

and it will have the following initial conditions.

$y(0) = y_0$  Initial displacement from the at rest, equilibrium position.

$y'(0) = (y_0)'$  Initial velocity.

*restart;*

*with(DETools) :*

*eq1 := diff(y(t), t) = v(t);*

$$eq1 := \frac{d}{dt} y(t) = v(t) \quad (1)$$

*eq2 := diff(v(t), t) = -\left(\frac{k}{m}\right) \cdot y(t) - \left(\frac{b}{m}\right) \cdot v(t);*

$$eq2 := \frac{d}{dt} v(t) = -\frac{k y(t)}{m} - \frac{b v(t)}{m} \quad (2)$$

*sys := [eq1, eq2];*

$$sys := \left[ \frac{d}{dt} y(t) = v(t), \frac{d}{dt} v(t) = -\frac{k y(t)}{m} - \frac{b v(t)}{m} \right] \quad (3)$$

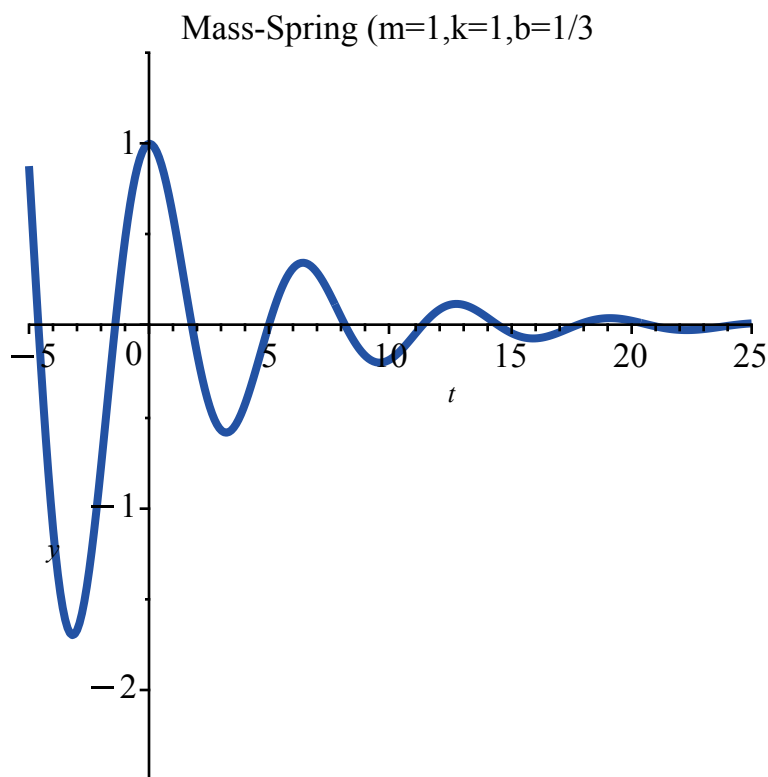
*sys1 := subs\left(b = \frac{1}{3}, m = 1, k = 1, sys\right);*

$$sys1 := \left[ \frac{d}{dt} y(t) = v(t), \frac{d}{dt} v(t) = -y(t) - \frac{v(t)}{3} \right] \quad (4)$$

*ics := [[0, 1, 0]];*

$$ics := [[0, 1, 0]] \quad (5)$$

```
DEplot(sys1, [y(t), v(t)], t=-5..25, ics, y=-2.5..1.5, title="Mass-Spring (m=1,k=1,b=1/3",
stepsize=0.1, scene=[t, y], size=[300, 300]);
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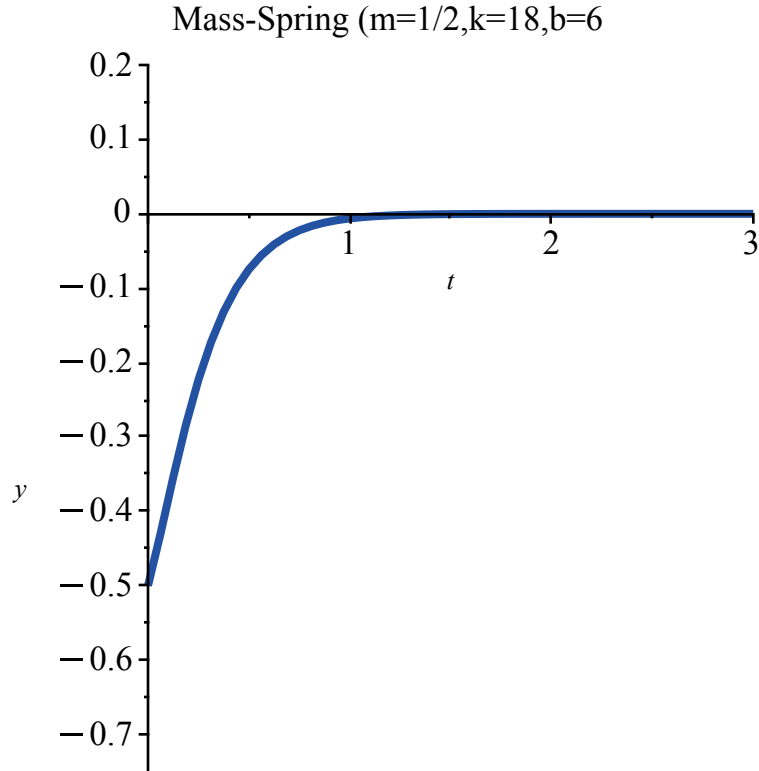


Now, let's put in different initial conditions to create a critically damped system.

```
sys2 := subs(b=6, m=1/2, k=18, sys); ics2 := [[0, -1/2, 1]]:
```

$$\text{sys2} := \left[ \frac{d}{dt} y(t) = v(t), \frac{d}{dt} v(t) = -36 y(t) - 12 v(t) \right] \quad (6)$$

`DEplot(sys2, [y(t), v(t)], t=0..3, ics2, y=-0.75..0.2, title="Mass-Spring (m=1/2,k=18,b=6",  
stepsize=0.1, scene=[t, y], size=[300, 300]);`



With Maple it is easy to see the solution but what does this equation look like.

First, explicitly define the initial conditions.

$$\begin{aligned} \text{inits} &:= y(0) = -0.5, v(0) = 1; \\ \text{inits} &:= y(0) = -0.5, v(0) = 1 \end{aligned} \quad (7)$$

Then, define the equations.

$$\begin{aligned} \text{eqs} &:= \frac{d}{dt} y(t) = v(t), \frac{d}{dt} v(t) = -36 y(t) - 12 v(t); \\ \text{eqs} &:= \frac{d}{dt} y(t) = v(t), \frac{d}{dt} v(t) = -36 y(t) - 12 v(t) \end{aligned} \quad (8)$$

Now solve the system.

$$dsolve(\{eqs, inits\}, \{y(t), v(t)\});$$

$$\left\{v(t) = -e^{-6t}(-12t - 1), y(t) = e^{-6t}\left(-2t - \frac{1}{2}\right)\right\} \quad (9)$$

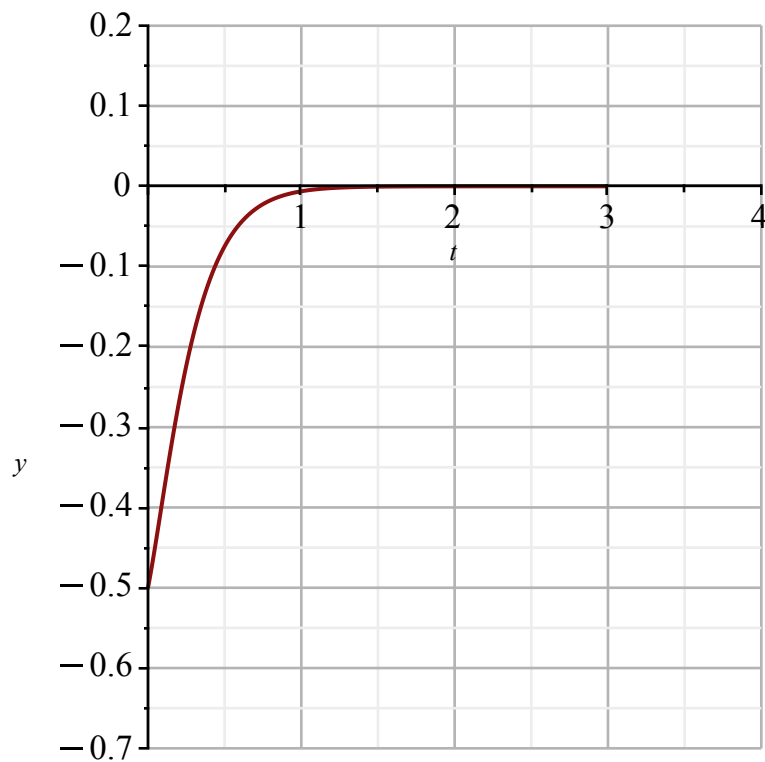
Here you can see what the position  $y(t)$  looks like.

Now let's plot it. First get a numeric solution to the system so you can plot  $y(t)$ .

$$sol\_numeric := dsolve(eval(\{eqs, inits\}), numeric);$$

$$sol\_numeric := \text{proc}(x\_rkf45) \dots \text{end proc} \quad (10)$$

$$plots:-odeplot(sol\_numeric, [t, y(t)], t=0..3, view=[0..4, -0.7..0.2], gridlines, size=[300, 300]);$$



Why is the above example referred to as critical damping. We need to look at our second order equation to find out.

$$m \cdot y'' + \gamma y' + k \cdot y = 0$$

Let's solve it.

$$f := y \mapsto m \cdot y^2 + \gamma \cdot y + k;$$

$$f := y \mapsto m \cdot y^2 + \gamma \cdot y + k \quad (11)$$

$$\text{solve}(f(y) = 0, y);$$

$$\frac{-\gamma + \sqrt{\gamma^2 - 4mk}}{2m}, -\frac{\gamma + \sqrt{\gamma^2 - 4mk}}{2m} \quad (12)$$

We see the expected result for the solution of a quadratic equation. What is interesting to us is what is happening under the square root sign. Three cases make sense.

1.  $\gamma^2 - 4mk$  is equal to zero.

For our examples above  $m$  is  $m$ ,  $b$  is  $\gamma$ , and  $k$  is  $k$ . So for the example immediately above  $m = 1/2$ ,  $k = 18$ ,  $b = 6$

$$m \quad (13)$$

$\sqrt{\gamma^2 - 4mk}$  is  $\sqrt{6^2 - 4 \cdot \left(\frac{1}{2}\right) \cdot 18}$  or  $\sqrt{36 - 36}$  or zero. So that is why the system was critically damped.

Note that since  $\gamma^2 - 4mk = 0$  then  $\gamma^2 = 4mk$  and  $\gamma = 2\sqrt{mk}$  which will be referred to as the critical damping coefficient.

$$\gamma^2 = 4mk \quad (14)$$

2.  $\gamma^2 - 4mk$  is greater than zero.

Here,  $\gamma$  will be greater than  $2\sqrt{mk}$  or the critical damping we defined in 1.

3.  $\gamma^2 - 4mk$  is less than zero.

Here,  $\gamma$  will be less than  $2\sqrt{mk}$  or the critical damping we defined in 1.

The problem set up for the above critically damped system started with a 16 lb object that stretches a spring 8/9 feet.

Add a damper that will exert a force of 12 lbs when the velocity is 2 ft/sec. Remember our equation is

$$m \cdot y'' + \gamma \cdot y' + k \cdot y = F(t) \text{ with } F(t) = 0.$$

So we have  $12 = \gamma \cdot 2 \Rightarrow \gamma = 6$ . (This was  $b$  in the example).

The mass is  $m = \frac{W}{g} = \frac{16}{32} = \frac{1}{2}$ .

Remember  $m \cdot g = k \cdot L$  so

$$k = \frac{(m \cdot g)}{L} = \frac{16}{\left(\frac{8}{9}\right)} = 18.$$

The spring is displaced 6 inches upwards and given an initial velocity of 1 ft/sec. Since downward is positive the initial conditions are

$$y(0) = -\frac{1}{2} \text{ and } (y(0))' = 1.$$

Now let's create an overdamped system. Let the damper exert a force of 17 lbs this time. Now we have  $17 = \gamma \cdot 2 \Rightarrow \gamma = 8.5$ . The  $\gamma$  is greater than the critical damping factor which was 6. Now we have

$$m \left( \frac{d^2}{dx^2} y(x) \right) + \gamma \left( \frac{d}{dx} y(x) \right) + k y(x) = 0$$

$$\frac{1}{2} \left( \frac{d^2}{dx^2} y(x) \right) + 8.5 \left( \frac{d}{dx} y(x) \right) + 18 y(x) = 0$$

Now let's multiply through by 2 and get

$$\left( \frac{d^2}{dx^2} y(x) \right) + 17 \left( \frac{d}{dx} y(x) \right) + 36 y(x) = 0$$

$$\frac{d^2}{dx^2} y(x) + 17 \frac{d}{dx} y(x) + 36 y(x) = 0 \quad (15)$$

$$f2 := y \mapsto y^2 + 17 \cdot y + 36 ;$$

$$f2 := y \mapsto y^2 + 17 \cdot y + 36 \quad (16)$$

$$\text{evalf}(\text{solve}(f2(y) = 0, y)) ;$$

$$-2.479202710, -14.52079729 \quad (17)$$

So there are 2 real roots.

Again, explicitly define the initial conditions.

$$\text{inits} := y(0) = -0.5, v(0) = 1 ;$$

$$\text{inits} := y(0) = -0.5, v(0) = 1 \quad (18)$$

Then, define the system equations.

$$\text{eqs2} := \frac{d}{dt} y(t) = v(t), \frac{d}{dt} v(t) = -17 \cdot v(t) - 36 y(t) ;$$

$$\text{eqs2} := \frac{d}{dt} y(t) = v(t), \frac{d}{dt} v(t) = -17 v(t) - 36 y(t) \quad (19)$$

Now solve the system.

$$\text{evalf}(\text{dsolve}(\{\text{eqs2}, \text{inits}\}, \{y(t), v(t)\}));$$

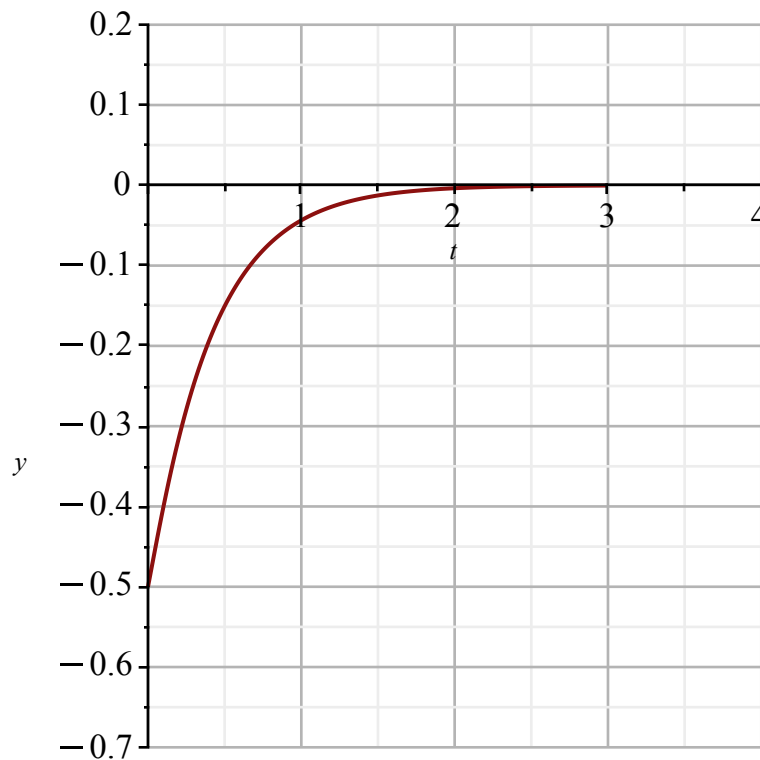
$$\{v(t) = 1.288932058 e^{-2.479202710 t} - 0.2889320583 e^{-14.52079729 t}, y(t) = -0.5198978095 e^{-2.479202710 t} + 0.0198978095 e^{-14.52079729 t}\} \quad (20)$$

Here you can see what the position  $y(t)$  looks like.



Now let's plot it. First get a numeric solution to the system so you can plot  $y(t)$  .

```
sol_numeric2 := dsolve( eval( {eqs2, inits} ), numeric );
sol_numeric2 := proc(x_rkf45) ... end proc (21)
plots:-odeplot(sol_numeric2, [t, y(t)], t=0..3, view=[0..4, -0.7..0.2], gridlines, size=[300, 300]);
```



In this case the overdamped case takes a little longer to settle to 0. Note that the critically damped system settles as quickly as possible without overshooting the steady state condition, 0 here. What does an underdamped system look like?

To create an underdamped system  $\gamma$  must be less than the critical value, 6 in these examples. Let the damper exert 5 lbs when the velocity is 2 ft/sec.

Now we have  $5 = \gamma \cdot 2 \Rightarrow \gamma = 2.5$ .

The  $\gamma$  is less than the critical damping factor which was 6. Now we have

$$m \left( \frac{d^2}{dx^2} y(x) \right) + \gamma \left( \frac{d}{dx} y(x) \right) + k y(x) = 0$$

$$\frac{1}{2} \left( \frac{d^2}{dx^2} y(x) \right) + 2.5 \left( \frac{d}{dx} y(x) \right) + 18 y(x) = 0$$

Now let's multiply through by 2 and get

$$\frac{d^2}{dx^2} y(x) + 5 \cdot \left( \frac{d}{dx} y(x) \right) + 36 y(x) = 0$$

$$f2 := y \mapsto y^2 + 5 \cdot y + 36 ;$$

$$f2 := y \mapsto y^2 + 5 \cdot y + 36 \quad (22)$$

$$\text{evalf}(\text{solve}(f2(y) = 0, y));$$

$$-2.500000000 + 5.454356055 I, -2.500000000 - 5.454356055 I \quad (23)$$

So there are 2 complex roots so we expect some sort of sinusoidal response.

Again, explicitly define the initial conditions.

$$\text{inits} := y(0) = -0.5, v(0) = 1;$$

$$\text{inits} := y(0) = -0.5, v(0) = 1 \quad (24)$$

Then, define the system equations.

$$\text{eqs3} := \frac{d}{dt} y(t) = v(t), \frac{d}{dt} v(t) = -5 \cdot v(t) - 36 y(t) ;$$

$$\text{eqs3} := \frac{d}{dt} y(t) = v(t), \frac{d}{dt} v(t) = -5 v(t) - 36 y(t) \quad (25)$$

Now solve the system.

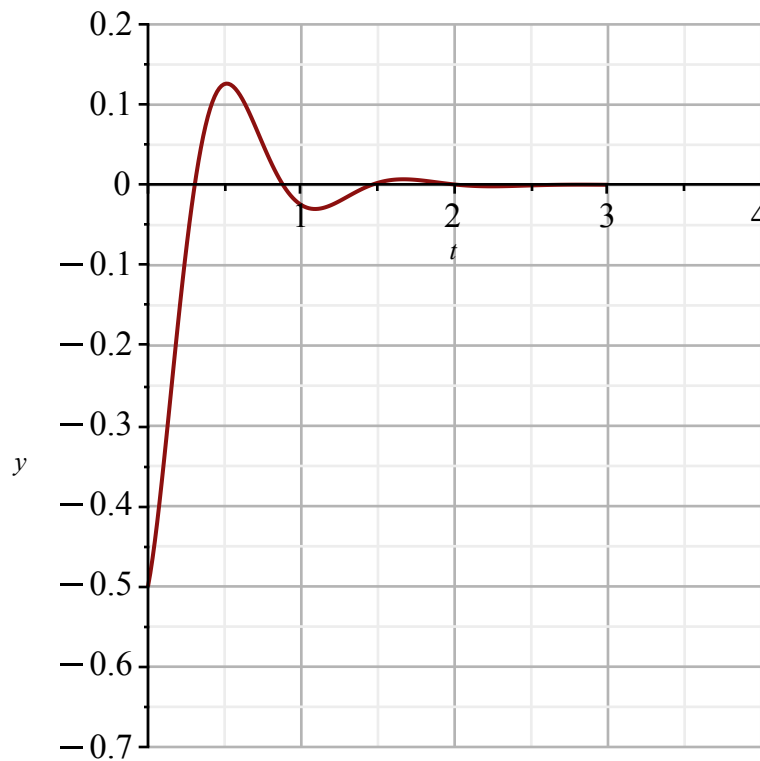
$$\text{evalf}(\text{dsolve}(\{\text{eqs3}, \text{inits}\}, \{y(t), v(t)\}));$$

$$\begin{aligned} \{ & v(t) = -0.5000000000 e^{-2.500000000 t} (-5.683530680 \sin(5.454356055 t) \\ & - 2. \cos(5.454356055 t)), y(t) = e^{-2.500000000 t} (-0.04583492483 \sin(5.454356055 t) \\ & - 0.5000000000 \cos(5.454356055 t)) \} \end{aligned} \quad (26)$$

Here you can see what the position  $y(t)$  looks like.

Now let's plot it. First get a numeric solution to the system so you can plot  $y(t)$  .

```
sol_numeric3 := dsolve( eval( {eqs3, inits} ), numeric );  
sol_numeric3 := proc(x_rkf45) ... end proc (27)  
plots:-odeplot(sol_numeric3, [t, y(t)], t=0..3, view=[0..4, -0.7..0.2], gridlines, size=[300,  
300]);
```



The overshoot is clearly seen in this latest plot of an underdamped system. |